

## ARTICLE

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# Cooling-by-measurement and mechanical state tomography via pulsed optomechanics

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Observing a physical quantity without disturbing it is a key capability for the control of individual quantum systems. Such back-action-evading or quantum non-demolition measurements were first introduced in the 1970s for gravitational wave detection, and now such techniques are an indispensable tool throughout quantum science. Here we perform measurements of the position of a mechanical oscillator using pulses of light with a duration much shorter than a period of mechanical motion. Utilizing this back-action-evading interaction, we demonstrate state preparation and full state tomography of the mechanical motional state. We have reconstructed states with a position uncertainty reduced to 19 pm, limited by the quantum fluctuations of the optical pulse, and we have performed 'cooling-by-measurement' to reduce the mechanical mode temperature from an initial 1,100 to 16 K. Future improvements to this technique will allow for quantum squeezing of mechanical motion, even from room temperature, and reconstruction of non-classical states exhibiting negative phase-space quasi-probability.

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xperiments are now beginning to investigate non-classical motion of massive mechanical devices<sup>1-3</sup>. This opens up new perspectives for quantum-physics-enhanced applications and for tests of the foundations of physics. A versatile approach to manipulate mechanical states of motion is provided by the interaction with electromagnetic radiation, typically confined to microwave or optical cavities. Such cavityoptomechanics experiments<sup>4-8</sup> have thus far largely concentrated on high-sensitivity continuous monitoring of the mechanical position<sup>9-14</sup>. Because of the back-action imparted by the probe onto the measured object, the precision of such a measurement is fundamentally constrained by the standard quantum limit (SQL)<sup>15,16</sup>, and therefore only allows for classical phase-space reconstruction<sup>9,17,18</sup>. In order to observe quantum mechanical features that are smaller than the mechanical zero-point motion, back-action-evading measurement techniques that can surpass the SQL<sup>19-22</sup> are required. Following the early insights of Braginsky and Khalili<sup>15</sup>, beating the SQL 'can be achieved only in one way: design the probe so it 'sees' only the measured observable'. Such back-action-evading techniques were first realized for the detection of optical quadratures<sup>23-25</sup> and have now also been used for precision measurement of atomic ensemble spin<sup>26-30</sup> and quantum non-demolition microwave photon counting<sup>31</sup>. In optomechanics, to perform a back-actionevading measurement of the mechanical position, a timedependent measurement scheme is required. One prominent example is the so-called 'two-tone approach'22,32, which uses a probe with an intensity that oscillates at twice the mechanical frequency. The field probes the mechanics periodically and the back-action imparted to the mechanical motion by the optical probe does not affect the measurement of the mechanical amplitude. This is closely analogous to a stroboscopic measurement of the mechanical motion<sup>22</sup>. Using the two-tone approach with a microwave probe field, a back-action-evading interaction was recently realized to measure a single quadrature of a nanomechanical resonator<sup>33</sup>.

Here we take a different tack to perform position measurements of a mechanical oscillator using single optical pulses. Our experimental approach employs optical pulses that have a duration much shorter than a mechanical period of motion. This provides a back-action-evading interaction for measuring the mechanical position because the interaction leaves the position unchanged, perturbing only the mechanical momentum, and was first suggested by Braginsky et al.<sup>34</sup> The precision of this pulsed measurement process is no longer limited by the SQL but is ultimately limited by the quantum optical phase noise. We implement a pulsed protocol<sup>35</sup>, where one or two pulses are used to prepare a motional state 'by measurement' and then a subsequent pulse is used for state tomography. Mechanical state preparation 'by measurement' is achieved by utilizing the information gained from the pulsed measurement to update the probability distribution that describes the motional state. The experiments reported here have been performed in the weak interaction regime, where the backaction itself is negligible; however, the pulsed measurements have a dramatic effect on the mechanical thermal state and the measurement precision we achieved was limited by the quantum optical phase noise. We therefore require a quantized description of the optical field; however, it is important to note that at this stage all the mechanical motional states presented here are classical, that is, they can be described by an incoherent mixture of mechanical coherent states. Our protocol can be used to prepare mechanical states independent of the initial mechanical thermal occupation and thus, no initial cooling of the mechanical motion is required. Moreover, by contrast to continuous schemes, our pulsed protocol has considerable resilience against the surrounding

mechanical thermal bath, as it can be performed on short time scales<sup>35</sup>. Employing our pulsed approach, mechanical dynamics rather than the steady-state can be conveniently probed and non-equilibrium mechanical behaviour can be characterized. Also note that pulsed quantum optomechanics operates fully in the so-called 'non-resolved sideband regime', in which the cavity decay rate is much larger than the mechanical frequency. Indeed, all results reported here were obtained without the use of an optical cavity.

#### Results

Experimental protocol. Our experimental setup is shown schematically in Fig. 1a. Optical pulses are injected into a Mach-Zehnder interferometer that has a micromechanical oscillating mirror in one of the two interferometer paths. The pulses are first divided by a beam-splitter that forms one intense beam that acts as a local oscillator (LO) and one weak beam that we will henceforth refer to as the signal. The signal is focussed onto and reflects from a micromechanical oscillator (Fig. 1b). During the reflection of the short optical pulse, changes to the position of the mechanical oscillator are negligible. The coherent optical pulse gains a phase shift in proportion to the mechanical position, which is accurately described by a phase quadrature displacement, as the mechanical position fluctuations are small. Concurrently, the radiation-pressure force of the reflection imparts momentum to the mechanical resonator. This momentum can be decomposed into a classical component due to the mean photon number and a component dependent upon the photon number fluctuations. Quantitatively, this optomechanical interaction is described by the input-output relations:

$$\begin{aligned} X_{\rm L}^{\rm out} &= X_{\rm L}^{\rm in}, \quad P_{\rm L}^{\rm out} = P_{\rm L}^{\rm in} + \chi X_{\rm M}^{\rm in}, \\ X_{\rm M}^{\rm out} &= X_{\rm M}^{\rm in}, \quad P_{\rm M}^{\rm out} = P_{\rm M}^{\rm in} + \chi X_{\rm L}^{\rm in} + \Omega \end{aligned} \tag{1}$$

Here, the subscripts label the light (L) and mechanics (M); X and P are the dimensionless amplitude (position) and phase (momentum) quadratures for the light (mechanics);  $\chi =$  $4\pi x_0 \sqrt{N}/\lambda$  quantifies the quadrature information exchanged between the light and the mechanics and determines the strength of the mechanical position measurement and  $\Omega = 8\pi x_0 N/\lambda$  is the classical momentum transfer to the mechanical oscillator (*N*, mean photon number per pulse;  $\lambda$ , optical wavelength;  $x_0 = (\hbar/2m_{\rm eff}\omega_{\rm M})^{1/2}$ , mechanical ground state width;  $m_{\rm eff}$ mechanical effective mass; and  $\omega_{\mathrm{M}}$ , mechanical angular frequency). After the optomechanical reflection, the signal then overlaps and interferes with the LO pulse on a 50/50 beamsplitter, where the (mean) phase between the LO and signal beams is set to be  $\pi/2$ . The intensities of both beam-splitter outputs are measured by photodiodes, and the photocurrents are subtracted to implement homodyne detection of the optical phase quadrature. A typical difference current time trace is shown in Fig. 1c, where the measurement outcome  $P_{\rm L}$  is the time integral over the pulse duration of the difference current.

After the pulsed measurement, the mechanical state of motion is changed as our knowledge of the mechanical position has increased. For an initial thermal state of the mechanical resonator with a large thermal occupation, that is,  $\chi^2(1+2\bar{n}) > 1$ , the means and variances of the mechanical quadratures, upon obtaining the measurement outcome  $P_{\rm L}$  are as follows<sup>35</sup>:

$$\langle X_{\rm M}^{\rm out} \rangle \simeq P_{\rm L}/\chi, \quad \langle P_{\rm M}^{\rm out} \rangle = \Omega,$$
  
 $\sigma_{X_{\rm M}^{\rm out}}^2 \simeq 1/(2\chi^2), \quad \sigma_{P_{\rm M}^{\rm out}}^2 = (\chi^2 + 1 + 2\bar{n})/2, \qquad (2)$ 

where  $\bar{n} \simeq k_{\rm B} T / \hbar \omega_{\rm M}$  is the mean occupation of the mechanical mode when in thermal equilibrium with the environment at temperature  $T > \hbar \omega_{\rm M} / k_{\rm B}$ . Notably the information gained from



**Figure 1 | Experimental setup.** (a) Schematic of the experimental setup used to perform state tomography and state preparation of the motional state of a mechanical resonator. In addition to the optical pulses, a weak continuous field is used to stabilize the interferometer phase using the homodyne output passed through a low-pass filter with cutoff frequency below the mechanical frequency. (b) Colourized optical micrograph of the high-reflectivity micro-mechanical cantilever fabricated for this experiment. The head of the cantilever, where the signal beam is focussed, is 100  $\mu$ m in diameter. (c) Example time trace of the homodyne output for a pair of 4  $\mu$ s pulses. (For clarity, the pulse rising and falling edges are not shown.) The measurement outcome  $P_L$  is the time integral of the homodyne output (indicated by the shaded region). Time resolved optical quantum noise is visible during the pulse.

the measurement reduces the mechanical position variance from  $\bar{n}$  to  $1/(2\chi^2)$ , which does not depend on the initial occupation. The resultant state of mechanical motion, following such a measurement, is no longer in thermal equilibrium with the surrounding environment and has a reduced effective thermal occupation  $\bar{n}_{\rm eff} = (\sigma_{X_{\rm M}}^2 \sigma_{P_{\rm M}}^2)^{1/2} - 1/2 \simeq (\bar{n}/(2\chi^2))^{1/2}$ . Moreover, a subsequent pulse performed after one quarter of a period of mechanical harmonic evolution can measure the mechanical momentum at the time of the first pulse to further reduce the effective occupation. This 'cooling-by-measurement' method for entropy reduction, that is, obtaining mechanical position and then momentum information on the initial state, is rapid and has considerable tolerance to both the initial thermal occupation and the surrounding thermal bath<sup>35</sup>. With future experimental improvements, this scheme allows for the generation of high purity and quantum-squeezed states of mechanical motion 'by measurement'. Owing to the resilience against mechanical thermal noise, this scheme may provide a more feasible route to quantum squeezing than parametric modulation<sup>9,17</sup>, which can be combined with continuous measurement and feedback<sup>36</sup>.

In our experiment, one or two pulses are used to prepare a mechanical state at a known time. Then a read-out pulse is made after time  $\theta/\omega_{\rm M}$  of mechanical harmonic evolution to sample the mechanical probability distribution of the  $\theta$ -rotated quadrature, that is, a marginal. Repeating this process many times and obtaining the marginals for a large number of mechanical phase-space angles  $\theta$  is sufficient to uniquely determine the mechanical quantum state of motion<sup>37</sup>. Quantum state tomography by measurement of the marginals was first realized with optical fields using homodyne interferometry<sup>38</sup> and has now become an indispensable tool in the field of quantum optics<sup>39</sup> being applied to other physical systems such as molecular vibration<sup>40</sup>, spin ensembles<sup>41</sup> and microwave fields<sup>42</sup>. Here we implement such mechanical state tomography by utilizing the pulsed measurement outcome probability distribution  $Pr(P_L) = \int dX_M \pi^{-1/2} \exp[-(P_L - \chi X_M)^2] Pr(X_M, \theta)$  that contains the mechanical marginals  $Pr(X_M, \theta) = \langle X_M | \rho_M^{in}(\theta) | X_M \rangle$ , where  $\rho_M^{in}(\theta)$  is the mechanical input state to be reconstructed after time  $\theta/\omega_M$  of harmonic evolution. In this experiment, we prepare and reconstruct mechanical motional states with features that are not smaller than  $\chi^{-1}$  and hence, unless otherwise noted, we use the optical measurement outcome distribution as an approximation for the mechanical distribution using the scaled outcome  $P_L/\chi$ .

Mechanical state preparation and reconstruction. The mechanical resonator used for this experiment is a micro-mirror cantilever constructed from an epitaxial  $Al_xGa_{1-x}As$  crystalline multilayer, see Fig. 1b. The use of such a monocrystalline material structure allows for a significant reduction of the mechanical damping of the resonator when compared with dielectric reflectors<sup>43</sup> and simultaneously provides high optical reflectivity. The crystalline material used here is nominally identical in composition and individual layer thickness to structures used previously<sup>44</sup> and is designed for maximum reflectivity at our

optical wavelength of 1,064 nm. The multilayer Bragg mirror comprises 40.5 layer pairs in order to minimize transmission losses. The cantilever was etched from a 6.88-µm thick multilayer and is 1.45 mm in length with a cantilever arm 5 µm in width with a circular head 100 µm in diameter, where the optical signal beam is focussed. For details of the microfabrication procedure see Cole<sup>45</sup>. Note that the resonator is etched directly from the multilayer mirror material and is therefore equally reflective at all points along the structure with an (intensity) reflectivity of 99.982%. This cantilever has a fundamental out-of-plane vibrational mode with frequency  $\omega_M/2\pi = 984.3$  Hz, effective mass  $m_{\rm eff} = 260$  ng (see the methods section), ground state width  $x_0 = 5.7 \times 10^{-15}$  m and a mechanical quality of  $Q = 3.1 \times 10^4$  in vacuum ( $10^{-5}$  mbar) and at room temperature measured via mechanical ringdown.

Our optical setup (Fig. 1a) was constructed from optical-fibrebased components that provides good phase stability and excellent spatial mode matching. Indeed, when the optical powers in the two arms of the interferometer are balanced, we observed an interference visibility exceeding 99.9%. We use a continuous laser source and generate optical pulses of duration 1 µs (excluding the pulse edges) with a fibre-based intensity modulator. The mean photon number in a signal pulse was up to  $10^7$  and in order to provide a homodyne signal well above the electronic noise, we use a large LO to signal ratio with up to 10<sup>10</sup> photons per LO pulse. (These photon numbers were determined via optical power measurement during continuous wave operation.) The signal pulses are directed onto the cantilever head using an antireflection-coated fibre focuser and are then retroreflected. To calibrate the proportionality between the measurement outcomes and the mechanical position, we reflect the signal beam from a rigid mirror adjacent to the mechanical resonator and scan the mirror position using a calibrated piezoelectric actuator, recording both the piezo scan positions and pulse measurement outcomes (see the methods section). For our mechanical resonator ground state width ( $x_0 = 5.7 \times 10^{-15}$  m), this photon number per pulse yields a measurement strength  $\chi$  of order  $10^{-4}$  and a momentum transfer  $\Omega$  of order unity. The radiation pressure backaction from the reflection of the pulse is smaller than the mechanical thermal noise and is not observed; however, as will be detailed in the following, this measurement strength has a strong effect on the mechanical thermal noise.

After a pulsed measurement is performed to sample a mechanical marginal, the mechanical state is reinitialized by first allowing it to return to equilibrium with the environment and then the mechanical state is reprepared. This process is repeated many times to accumulate sufficient data to characterize the statistical properties of the mechanical motion. The marginal distributions were then obtained by constructing a histogram from the many measurement outcomes recorded for each mechanical phase-space angle  $\theta$ . As the states studied here are symmetric about the  $X_{\rm M}$  and  $P_{\rm M}$  axes, we measure a set of many marginals with angles between  $\theta = 0$  and  $\theta = \pi/2$  to fully characterize the state of motion. The phase-space probability distribution  $W(X_{\rm M}, P_{\rm M})$  is then obtained by using the inverse Radon transformation on the set of marginals.

The measurement results we obtained for motional state preparation and reconstruction are summarized in Fig. 2. In Fig. 2a a reconstruction of an initial thermal state that is driven by white noise up to a mode temperature of 1,100 K that has width  $\sigma_x = 1.2$  nm is shown. This temperature was obtained using the equipartition theorem  $k_{\rm B}T_{\rm eff} = m_{\rm eff}\omega_{\rm M}^2\sigma_x^2$ , where the mechanical position variance  $\sigma_x^2$  was obtained from the calibrated measurement outcome distribution after subtracting the optical noise contribution. A single pulsed measurement made on this initial thermal state generates a motional state that has a reduced position uncertainty (Fig. 2b). The observed momentum distribution of this state, however, is unchanged as the back-action to the mechanical momentum made by the reflection of the optical pulse is much smaller than the mechanical thermal noise. Each pulsed measurement generates a mechanical state with a random but known mean due to the random measurement outcome, see Equation (2). By making the transformation  $P_{\rm L}^{(r)} \rightarrow P_{\rm L}^{(r)} - P_{\rm L}^{(p1)}\cos\theta$ , where the superscripts (*r*) and (*p*<sub>1</sub>) indicate read-out and preparation, respectively, this random mean is subtracted and the distribution of the mechanical state can be characterized. We would like to emphasize here that no 'post selection' is performed and all measurement outcomes are used in this process. Furthermore, our experimental pulsed technique demonstrates the back-action-evading feature of measurement repeatability, that is, a subsequent measurement is not affected by a prior measurement<sup>19-22</sup>. Specifically, in our case the measurement results of the read-out pulse made a short time after the preparation pulse are the same as the preparation pulse to within the optical quantum noise. The plots for Fig. 2a,b were generated from the same data set, where the statistics of the preparation pulse alone characterizes the unconditional initial thermal state and the read-out pulse characterizes the conditional mechanical state. A 1,100 K thermal state (which has a root-mean-square (RMS) amplitude less than a factor of two larger than a thermal state at 300 K) was used to increase the mechanical contribution to the optical phase noise over the relevant  $\sim$  DC to MHz bandwidth for our pulses to improve the signal-to-noise ratio for mechanical conditional state preparation.

In Fig. 2c the reconstruction of a mechanical state of motion prepared via two pulsed measurements separated by one quarter of a mechanical period is shown. The width of the mechanical phase-space distribution has been significantly reduced in both the position and momentum quadratures compared with the initial thermal state (Fig. 2a) and hence the effective mode temperature has significantly decreased. This method of cooling is rapid as it takes place well within a single mechanical period and is, to the best of our knowledge, yet to be experimentally reported elsewhere. For this pulse sequence the read-out pulse outcome is transformed using  $P_{i}^{(r)} \rightarrow P_{i}^{(r)} - P_{i}^{(p2)} \cos \theta + P_{i}^{(p1)} \sin \theta$ , where  $\theta$  is the angle of mechanical evolution made between the second preparation pulse and the read-out pulse. Ideally, for this mechanical state, the width of the mechanical marginals should be constant for all  $\theta$ ; however, in our experiment the phase correlation between the pulses reduces with increasing pulse separation as low frequency noise, due to imperfect phase locking, enters the signal. This results in a broadening of the conditional mechanical marginals as  $\theta$  increases. The effective temperature  $T_{\rm eff} = m_{\rm eff} \omega_{\rm M}^2 \sigma_x^{(\theta=0)} \sigma_x^{(\theta=\pi/2)} / k_{\rm B}$  observed for this state is 16 K, which depends on the product of the standard deviations of the position and momentum quadratures. Were the pulses to remain correlated to within the quantum noise, the effective temperature that could be reached for this measurement strength, taking the effects of mechanical rethermalization into account, would be 4.4 K (ref. 35). We would like to highlight here that rethermalization contributes to less than 1% of this value. To summarize the observed effects of single- and two-pulse mechanical state preparation discussed above, Fig. 2e provides a plot of the measured mechanical widths with  $\theta$  for the initial thermal state and the two mechanical conditional states. In this plot, the mechanical widths were determined from the calibrated pulse outcome distributions after subtracting the optical noise contributions that were measured independently. The data for both of the mechanical conditional states were taken with the same signal pulse powers and for each phasespace angle 300 pulses were recorded to construct the histograms.



Figure 2 | Mechanical motional state preparation and full state reconstruction using optical pulsed quantum measurement. The uppermost row shows the pulse protocols (pink - preparation, red - tomography). The two rows below show a subset of the measured probability distributions of the mechanical quadratures  $Pr(x,\theta)$  and the reconstructed phase-space distributions  $W(X_M,P_M)$ , respectively. The phase-space distributions were reconstructed using nine marginal angles up to  $\theta \cdot \pi/180 = 90^{\circ}$  (with a larger number of bins used than that shown for the marginals). For our current measurement strength, that is,  $\chi$  < 1, all the mechanical motional states reconstructed here can be described classically. (a) In the first column, tomography and reconstruction of an initial mechanical thermal state driven by white noise up to a mode temperature of 1100 K is shown. The red dashed circle has a radius equal to  $2\sigma$  of the initial thermal distribution. (b) A single pulsed measurement reduces the mechanical position variance, but leaves the momentum distribution unchanged. (c) 'Cooling-by-measurement' performed with two pulses separated by one quarter of a mechanical period rapidly reduces the mechanical state's entropy. The effective temperature of the mechanical state reconstructed here has been reduced to 16 K. (d) State reconstruction of a non-Gaussian mechanical state of motion generated by resonant sinusoidal drive. (e) The (one s.d.) width of the position distribution observed for states (**a-c**) with phase-space angle  $\theta$ . The thermal state (red points) shows a position width approximately twice of that when at room temperature (dashed line). State (b) has a reduced position width for small phase-space angles (purple points). The position width of state (c) is reduced for all phase-space angles (blue points). The solid lines are theoretical fits obtained using Equation (2) generalized for all  $\theta$  as well the two-pulse-preparation case. (f) Plot of the conditional mechanical width with pulse strength obtained using two pulses separated by 5° of mechanical evolution. The dashed line is a theoretical fit with a model using two units of optical quantum noise and finite mechanical evolution. The solid line is the inferred conditional mechanical width immediately after the preparation pulse. The vertical line indicates the pulse strength used for states (a-c). The error bars on (e) and (f) indicate a one s.d. uncertainty.

As an example of a non-Gaussian state of motion, we have reconstructed a driven thermal state (Fig. 2d) that was generated by applying a sinusoidal drive on resonance with the mechanical eigenfrequency. Note that the two peaks in the mechanical marginals are narrower than the broad thermal state in Fig. 2a, as this state was prepared at room temperature without the white noise drive. Even though this state of motion and the thermal state are rotationally invariant in phase space, many marginals are measured for their reconstruction. On the other hand, the conditional mechanical states of motion are not rotationally invariant in phase-space as the time of the preparation pulse(s) sets the time for  $\theta = 0$ . Note that this pulse-based tomography scheme does not measure the angle  $\theta = 0$  as the read-out pulse is temporally separated from the preparation pulse(s). The lack of this marginal angle causes the rippling near  $X_M = 0$  in the reconstructed phase-space distributions. By employing shorter pulses and measuring the marginals at smaller angles this rippling can be reduced.

To demonstrate the scaling of our measurement strength in Fig. 2f, the conditional mechanical width observed by a read-out pulse made after 5° of mechanical free evolution is plotted with increasing pulse amplitude. For this pulse separation, the two pulses are well correlated and the width of the conditional mechanical state is limited by the optical quantum noise in the measurement (see the methods section for more details). As the signal pulse strength is increased, the standard deviation of the conditional mechanical position distribution decreases with  $N^{-1/2}$ , which is a result of the optical number-phase uncertainty relation. The dashed line in the plot is a theoretical prediction including the two units of optical shot noise, one each for the preparation and read-out pulses, and the small contribution from the mechanical evolution between the two pulses. The relative amplitudes for the data points were measured precisely and scaled by a free fitting parameter into units of square-root photon number, where the photon number per pulse obtained is consistent with measurements of the optical power made during continuous wave operation. For the largest optical pulse strength used the statistics of the read-out pulse demonstrate a conditional mechanical width (after the preparation pulse) of  $\sigma_x = 19$  pm corresponding to a measurement strength of  $\chi = 2.1 \times 10^{-4}$ .

#### Discussion

The techniques developed in this work provide the ability to experimentally perform quantum optomechanics in the time domain. This offers significant potential for optomechanics-based quantum information and quantum metrology applications by providing the framework for quantum state preparation of a mechanical resonator via quantum measurement<sup>46</sup>. One may then also envision combining such measurement based state preparation with feedback to implement full quantum control<sup>47</sup>. One exciting example of mechanical dynamics that can be probed by pulsed optomechanics has been recently theoretically discussed by Buchmann et al.48, where pulsed measurements, as now realized in this work, are considered for the observation of quantum tunnelling of a mechanical oscillator in a double-well potential. Another example for quantum state preparation is that, even though the optomechanical interaction used here is linear with the mechanical position, by exploiting the optical nonlinearity,  $X_{\rm M}^2$  measurements with a strength significantly larger than that attainable with dispersive optomechanics can be performed<sup>49</sup>. An  $X_M^2$  measurement can be used to conditionally prepare highly non-Guassian mechanical superposition states and experimentally characterizing the decoherence of such states is important to determine the feasibility of using mechanical elements for coherent quantum applications and can also be used to empirically test collapse models<sup>50-53</sup>. The pulsed measurements performed here may also be utilized for a quantum non-demolition measurement-based light-mechanics quantum interface<sup>54</sup>. Furthermore, a sequence of four pulsed optomechanical interactions can be used to generate non-classical mechanical states of motion via an optomechanical geometric phase<sup>55</sup> and can even be used to experimentally explore potential quantum-gravitational phenomena<sup>56</sup>.

For this experiment, to prepare a quantum-squeezed state of mechanical motion, the measurement strength needs to be increased to  $\chi > 1$ . An effective route to meet this requirement would be to employ an optical cavity to enhance the optomechanical interaction. Using the experimental parameters achieved in this work, a cavity finesse of  $10^4$  is sufficient. As such a cavity simultaneously requires a high finesse, as well as a large bandwidth to accommodate a short optical pulse, this is best achieved with an optomechanical microcavity<sup>35</sup>. Such improvements to the measurement sensitivity will not only enable

Wigner reconstruction with significant negativity but, owing to this pulsed protocol's resilience against mechanical thermal noise, may also allow the generation of non-classical mechanical states in the regime of room temperature quantum optomechanics.

#### Methods

Verification of optical quantum noise. To verify that the measurement scheme used here is optical quantum noise limited, we measured the phase quadrature conditional variance of a pair of optical pulses with increasing total photon number, that is, the sum of the signal and LO photons per pulse, while keeping the signal to LO ratio fixed, see Fig. 3. As with our calibration procedure, the signal beam is focussed onto a rigid mirror adjacent to the mechanical oscillator to prevent coupling to the mechanical motion. The pulse separation used for this measurement was 14.1  $\mu$ s, which would correspond to 5° of mechanical free evolution, and is the same as that used for the data set shown in Fig. 2f. With this pulse separation the conditioning is essentially the second pulse outcome minus the first pulse outcome. The quantum noise components of these two temporal modes are uncorrelated; however, the lower frequency classical noise components vary slowly between the two pulses and are thus suppressed by the conditioning. Quantum mechanics predicts a linear dependence for the variance with total photon number; however, had classical phase noise been the dominant contribution, a quadratic dependence with the total photon number per pulse would have been observed. (For a discussion on experimental aspects of observing optical quantum noise, see Bachor and Ralph<sup>57</sup>). During this measurement, we were limited to a total photon number of  $10^{10}$  as the phase lock performance dramatically reduced beyond this point. Were we able to measure beyond this optical power the classical phase noise would have eventually become the dominant noise and the conditional mechanical variance that could have been achieved would have saturated.

The data points for Fig. 3 were obtained from Gaussian fits to histograms of the conditional outcomes. The error bars indicate a one s.d. uncertainty as determined from the fit. The observed conditional variance shows a linear dependence with the total photon number with a 'goodness of fit' parameter  $R^2 = 0.97$ , taking the error bars into account. This demonstrates that, up to a total photon number of order  $10^{10}$ , the conditional variance is quantum noise limited.

Also included in Fig. 3 is the measured electronic noise, that is, the conditional variance observed using no light. This contribution is 19.5 dB smaller than the observed optical quantum noise at the data point with the highest optical intensity  $(N_{\text{TOT}} = 9.5 \times 10^9)$ .

**Effective mass measurement.** The optically probed effective mass of a mechanical vibrational mode depends upon the geometry and material properties of the mechanical structure as well as the intensity profile of the incident optical beam. The mass associated with the mechanical displacement mode shape, that is, the modal mass, is in general less than the total mass of the structure; however, the optically probed effective mass can have a strong dependence on the position and profile of the optical beam. We estimate the optically probed effective mass of the cantilever in our experiment using a combination of measurements and finite element analysis. Using the established values for the relevant elastic constants averaged over the crystalline multilayer ( $C_{11} = 119.6$ ,  $C_{12} = 55.5$ ,  $C_{44} = 59.1$  GPa) and the average material density 4,476 kg m<sup>-3</sup>, the lateral geometry of the



**Figure 3 | Verification of optical quantum noise.** The measured optical phase quadrature conditional variance  $\sigma_{P_L^{(2)}|P_L^{(1)}}^2$  plotted with the total photon number (signal plus LO) per pulse  $N_{\text{TOT}}$ . The linear dependence observed demonstrates that the measurement scheme is optical quantum noise limited up to  $N_{\text{TOT}} \simeq 10^{10}$ . Were classical phase noise to be the dominant noise source, a quadratic dependence would be observed. Error bars indicate a one s.d. uncertainty.



Figure 4 | Optically probed effective mass. (a) Finite element simulations of the vibrational modes (no. 2, 4, 8 and 10) that have the four lowest optically probed effective masses. The displacements shown are exaggerated (blue to red indicates increasing displacement) and the outline indicates the cantilever rest position. The optically probed effective mass is given next to each mode number and underneath are the simulated mechanical frequencies in brackets and the measured frequencies. (b) Schematic of the cantilever and the focussed signal beam. The coordinate axis used when determining the optically probed effective mass is in the centre of the cantilever head on the mirror surface.

simulated resonator is adjusted until minimal error is found between the measured and simulated eigenfrequencies for the first four out-of-plane mechanical modes, see Fig.4a. (Note that the lowest frequency vibrational mode for our cantilever is an in-plane mode as the cantilever used is slightly thicker than wide.) A mean discrepancy between the measured and simulated frequencies of 6.1% was obtained by reducing the feature linewidth by 0.875 µm with respect to the lithographic mask. Note that the thickness of the free-standing mirror material was not used as a fitting parameter as it was accurately determined from the reflectance spectrum of the mirror<sup>58</sup> and found to be 6.88 µm. Once the geometry is determined, the effective mass is calculated via the volume integral<sup>59</sup>

$$m_{\rm eff} = \frac{\rho \int \int \int dx dy dz (u^2 + v^2 + w^2)}{D^2}$$
(3)

Here,  $\rho$  is the material density; u, v and w are the displacements of the body along the x, y, and z directions, respectively, and the optically probed displacement D is the overlap between the mechanical deflection and the optical Gaussian intensity profile, that is

$$D = \frac{1}{2\pi r_0^2} \int \int dx dy w(x, y, z = 0) \exp\left[-\frac{x^2 + y^2}{2r_0^2}\right]$$
(4)

where  $r_0$  is the standard deviation of the Gaussian optical intensity profile and the coordinate axis used for x, y and z has its origin in the centre of the cantilever head, see Fig. 4b. The antireflection-coated fibre focuser used in our experiment provides an optical beam diameter  $(4r_0)$  of 10.6 µm, which is much smaller than the nominal cantilever head diameter of  $100 \,\mu\text{m}$  (as fabricated diameter of  $98.25 \,\mu\text{m}$ ). Thus, for the fundamental out-of-plane mode, there is only a weak optical beam position and width dependence on the effective mass. (In this case, the effective mass is approximately equal to the intrinsic modal mass.) We have determined that the fundamental out-of-plane mechanical mode utilized in our experiment, which oscillates at 984.3 Hz, has an effective mass of 260 ng and thus a spring constant of 0.01 N m<sup>-1</sup>. For the higher order modes of the structure; however, lateral displacement of the beam leads to a rapid change in the effective mass. To minimize the contribution from these higher mechanical modes it is necessary to carefully position the optical beam. Assuming careful alignment, the geometry of our mechanical structure is such that the contributions from higher order modes are further suppressed as the effective mass rapidly increases with mode number. Indeed, the unconditional RMS amplitudes of modes no. 4, 8 and 10 are 2.4%, 0.4% and 0.1% that of mode no. 2, respectively.

**Calibration procedure.** We have used a two-step calibration procedure to determine the proportionality between the pulsed homodyne measurement outcomes and the mechanical displacement. During this procedure the signal beam is focused onto the chip edge, that is, a rigid unpatterned part of mirror material adjacent to the mechanical resonator, to prevent mechanical motion contributing to the signal. First, we calibrate the displacement of a piezoelectric actuator, which our fabricated structure containing the mechanical oscillator is placed upon, in response to a known drive voltage. We then drive the piezo and record the pulse measurement



**Figure 5 | Piezo calibration.** Plot of the homodyne output (red, left axis) during piezo drive (blue, right axis) to calibrate the piezo response. The sinusoidal drive generates a peak-to-peak piezo scan of one half of an optical wavelength that was determined by adjusting the drive amplitude until the turning points in the homodyne output coincide (dashed line).

outcomes during the controlled actuation in order to calibrate the pulsed interferometer. Each step is detailed below in the next two subsections, respectively.

Piezo calibration. To calibrate the piezoelectric actuator, we applied a sinusoidal drive voltage and used a continuous signal beam to monitor the piezo motion. The frequency of the drive was chosen such that the piezo mechanical response was either in or out of phase with the drive voltage. (Experimentally, care was needed to find a suitable drive frequency as the piezo does not have a flat spectral response.) During this procedure, the phase between the signal and LO beams does not require locking, and the piezo drive was at a higher frequency than the phase noise components in the interferometer. We then adjusted the drive amplitude such that the peak-to-peak piezo motion was one half of the optical wavelength. This can be done precisely as the difference current output of the interferometer has separate turning points occurring at the same level for this modulation depth and is then proportional to  $\cos[\varphi_0 + \pi \sin \omega t]$ , see Fig. 5, here  $\varphi_0$  is the (unlocked) slowly varying phase in the interferometer and  $\omega$  is the piezo drive angular frequency. As  $\varphi_0$  slowly changes this merely shifts the level of the turning points. In our experiment, we used a drive frequency of 1.06 kHz and exploited a resonance of the piezo to achieve a peak-to-peak scan of 532 nm using 4.6 Vpp.

**Pulse calibration**. Using the same piezo drive frequency as above, and using the piezo actuator calibration value (metres per Volt) obtained, the actuator was scanned with a reduced amplitude so that the optical phase shifts are small. (It was verified that the piezo responds linearly with the applied Voltage over our range of interest.) Then, during the piezo scan, pulsed position measurements are performed and both the voltage applied to the piezo at the time of the measurement and the pulsed measurement outcomes are recorded. The proportionality between these recorded values is used to obtain the outcome per metre calibration. This calibration value is optical amplitude dependent and had to be measured for several optical amplitudes for the measurement shown in Fig. 2f.

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#### Author contributions

All authors contributed to the experiment, the data analysis and writing of the manuscript.

#### Additional information

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